

Owner	Doc no-	Rev no	
	Project-	Date	
Client	Title	Qty	
	Tag no		
DESIGN FOR THE ANCHOR CHAIR			
(CHAIR DESIGN AS PER AISI E-1, VOLUME II, PART VII)			
Anchor Bolt Design Load ,	P	=	187.020000 Mpa
Provided Distance From Outside Of Top Plate To Edge Of Hole,	f	=	98.5 mm
Distance Between Vertical Palates,	g	=	100 mm
Anchor Bolt Diameter,	d	=	33 mm
Anchor Bolt Eccentricity,	e	=	108.5 mm
Top Plate Width Along The Shell,	a	=	150 mm
Anchor Chair Height,	h	=	275 mm
Radius Of Shell To The Centre Line Of Shell Plate,	R	=	5500 mm
Corroded Bottom Plate Thk,	m	=	6.5 mm
Top Plate Thickness	c	=	21.7 mm
Radius Of Shell	w	=	5500 mm
Bottom Shell Course Thickness (Corroded)	t	=	16.5 mm
Top plate length	b	=	225 mm
Anchor bolt diameter	d	=	33 mm
Vertical plate width (Average)	k	=	153 mm
Vertical plate thickness	j	=	14 mm
TOP PLATE DESIGN			
Critical stress in top plate , S	=	$P / (f \times c^2 \times ((0.375 \times g) - (0.22 \times d)))$	= 78.67 Mpa
		HENCE	S < 172 Mpa
			SAFE
Bending plus direct stress in shell, Sb	=	$P \times e / t^2 \times (1.32 \times Z / (1.43 \times a \times h^2 / (R \times t)) + (4 \times a \times h^2)^{0.333}) + 0.031 / (R \times t)^{0.5}$	= 123.29 Mpa
		HENCE	Sb < 172 Mpa
			SAFE
Where Reduction Factor	=	$1 / ((0.177 \times a \times m / (R \times t)^{0.5}) \times (m/t)^2 + 1)$	= 0.997
VERTICAL SIDE PLATE			
Minimum thickness is greater of 0.5 inch or 0.04 (h-c)	=	0.5	or 0.04 (h-c)
	=	0.5	or 0.40
	=	0.5	inch
Minimum thickness	=	12.7	mm
	j k	$\geq P / 25$	
	3.32	> 1.08	
	HENCE	J k > P / 25	SAFE
WELD SIZE CALCULATION			
W_v	=	$P / (a + 2h)$	= 6.8 Mpa
W_h	=	$P e / (ah + 0.667 h^2)$	= 5.6 Mpa
W	=	$(W_v^2 + W_h^2)^{0.5}$	= 8.8 Mpa
	HENCE	W 8.81 < 15.65 Mpa	SAFE