

Owner		Doc no-		Rev no	
		Project-		Date	
Client		Title		Qty	
		Tag no			

DESIGN FOR THE ANCHOR CHAIR

(CHAIR DESIGN AS PER AISI E-1, VOLUME II, PART VII)

Anchor Bolt Design Load ,	P	=	187.020000 Mpa	27.1 Ksi
Provided Distance From Outside Of Top Plate To Edge Of Hole,	f	=	98.5 mm	3.88 inch
Distance Between Vertical Palates,	g	=	100 mm	3.94 inch
Anchor Bolt Diameter,	d	=	33 mm	1.30 inch
Anchor Bolt Eccentricity,	e	=	108.5 mm	4.27 inch
Top Plate Width Along The Shell,	a	=	150 mm	5.91 inch
Anchor Chair Height,	h	=	275 mm	10.83 inch
Radius Of Shell To The Centre Line Of Shell Plate,	R	=	5500 mm	216.54 inch
Corroded Bottom Plate Thk,	m	=	6.5 mm	0.26 inch
Top Plate Thickness	c	=	21.7 mm	0.85 inch
Radius Of Shell	w	=	5500 mm	216.54 inch
Bottom Shell Course Thickness (Corroded)	t	=	16.5 mm	0.65 inch
Top plate length	b	=	225 mm	8.86 inch
Anchor bolt diameter	d	=	33 mm	1.30 inch
Vertical plate width ( Average )	k	=	153 mm	6.02 inch
Vertical plate thickness	j	=	14 mm	0.55 inch

TOP PLATE DESIGN

Critical stress in top plate , S	=	$P / (f \times c^2 \times ((0.375 \times g) - (0.22 \times d)))$	=	78.67 Mpa	11.41 ksi
HENCE S < 172 Mpa SAFE					
Bending plus direct stress in shell, Sb	=	$P \times e / t^2 \times (1.32 \times Z / (1.43 \times a \times h^2 / (R \times t) + (4 \times a \times h^2)^{0.333}) + 0.031 / (R \times t)^{0.5})$	=	123.29 Mpa	17.88 ksi
HENCE Sb < 172 Mpa SAFE					
Where Reduction Factor	=	$1 / ((0.177 \times a \times m / (R \times t)^{0.5}) \times (m / t)^2 + 1)$	=	0.997	

VERTICAL SIDE PLATE

Minimum thickness is greater of 0.5 inch or 0.04 (h - c)	=	0.5	or	0.04 ( h - c )
	=	0.5	or	0.40
	=	0.5		inch
Minimum thickness	=	12.7		mm
	j k	≥	P / 25	
	3.32	>	1.08	
HENCE J k > P / 25 SAFE				

WELD SIZE CALCULATION

W <sub>v</sub>	=	$P / (a + 2h)$	=	6.8 Mpa	0.98 Ksi
W <sub>H</sub>	=	$P \times e / (a \times h + 0.667 h^2)$	=	5.6 Mpa	0.82 Ksi
W	=	$(W_v^2 + W_H^2)^{0.5}$	=	8.8 Mpa	1.28 Ksi
HENCE W 8.81 < 15.65 Mpa SAFE					